

Dirac structures and Boundary Control Systems

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Introduction

Context

↪ Collaboration **LAGEP / University of Twente** (The Netherlands) within GEOPLEX () and Van Gogh projects.

↪ *Network perspective* on physical systems' theory for **modelling, simulation and control design**. Following the same idea behind the bond graph formalism (in finite dimension) :

- interconnection of small set of atomic elements with particular energetic behavior (storing, dissipation or conversion)
- interaction with the environment by means of a port
- interdomain coupling using a network structure called Dirac structure

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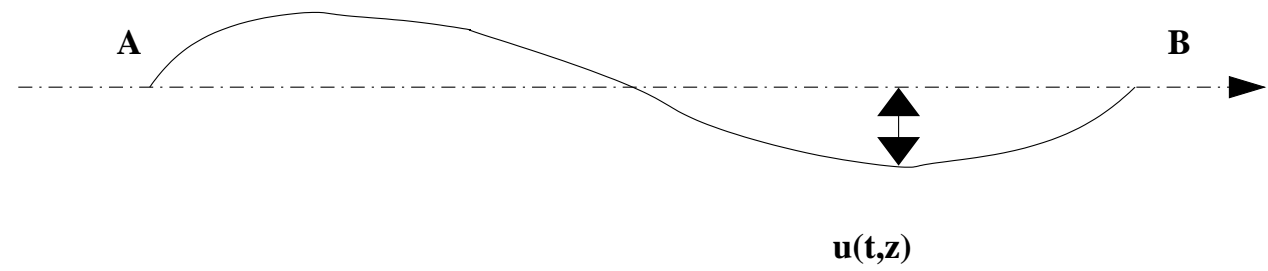
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↪ Systems described by PDE and **skew symmetric operators** : wave equations, beams (Bernuouilli, Timoshenko, ...). **Existence and nature of solutions** in function of the choice of the **boundary conditions**.

↪ **Vibrating string**



The classical modelling is based on the wave equation : Newton's law + Hooke's law (restoring force proportional to the deformation)

$$\frac{\partial^2 u(z, t)}{\partial t^2} = \sigma^2 \frac{\partial^2 u(z, t)}{\partial z^2} \quad \text{where} \quad |\sigma| = \sqrt{\frac{T(z)}{\mu(z)}}$$

The structure of the model is not apparent. How to choose the boundary conditions ???

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The energy and state variables

Let choose as state variables the energy variables:

- the strain $\epsilon = \frac{\partial u(z,t)}{\partial z}$

- the elastic momentum $p = \mu v(z,t)$

The total energy is given by : $H(\epsilon, p) = U(\epsilon) + K(p)$

- $U(\epsilon)$ is the elastic potential energy:

$$U(\epsilon) = \int_a^b \frac{1}{2} T(z) \left(\frac{\partial u(z,t)}{\partial z} \right)^2 = \int_a^b \frac{1}{2} T \epsilon(z,t)^2$$

where $T(z)$ denotes the elastic modulus.

- $K(p)$ is the kinetic co-energy:

$$K(p) = \int_a^b \frac{1}{2} \mu(z) v(z,t)^2 = \int_a^b \frac{1}{2} \frac{1}{\mu(z)} p^2(z,t)$$

where $\mu(z)$ denotes the string mass.

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Interdomain coupling

The vector of fluxes is given by:

$$\beta = \begin{pmatrix} v(t, z) \\ \sigma \end{pmatrix}$$

where $v(z, t)$ is the velocity and $\sigma = T\epsilon$ the stress.

The vector of fluxes β may be expressed in term of the generating forces :

$$\beta = \underbrace{\begin{pmatrix} 0 & 1 \\ 1 & 0 \end{pmatrix}}_{\text{canonical inerdomain coupling}} \underbrace{\begin{pmatrix} \frac{\delta H}{\delta \epsilon} \\ \frac{\delta H}{\delta p} \end{pmatrix}}_{\text{generating forces}}$$

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Hamiltonian formulation

From the conservation laws:

$$\frac{\partial}{\partial t} \begin{pmatrix} \epsilon \\ p \end{pmatrix} + \frac{\partial}{\partial z} \begin{pmatrix} v \\ \sigma \end{pmatrix} = 0$$

Consequently

$$\frac{\partial}{\partial t} \begin{pmatrix} \epsilon \\ p \end{pmatrix} = -\frac{\partial}{\partial z} \begin{pmatrix} 0 & 1 \\ 1 & 0 \end{pmatrix} \begin{pmatrix} \frac{\delta H}{\delta \epsilon} \\ \frac{\delta H}{\delta p} \end{pmatrix}$$

and

$$\frac{\partial}{\partial t} \begin{pmatrix} \epsilon \\ p \end{pmatrix} = \begin{pmatrix} 0 & -\frac{\partial}{\partial z} \\ -\frac{\partial}{\partial z} & 0 \end{pmatrix} \begin{pmatrix} \frac{\delta H}{\delta \epsilon} \\ \frac{\delta H}{\delta p} \end{pmatrix}$$

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Skew symmetry

$$\underbrace{\frac{\partial}{\partial t} \begin{pmatrix} \epsilon \\ p \end{pmatrix}}_f = \underbrace{\begin{pmatrix} 0 & -\frac{\partial}{\partial z} \\ -\frac{\partial}{\partial z} & 0 \end{pmatrix}}_{\mathcal{J} = \text{matrix}} \underbrace{\begin{pmatrix} T(z) & 0 \\ 0 & \frac{1}{\mu(z)} \end{pmatrix} \begin{pmatrix} \epsilon \\ p \end{pmatrix}}_{e = \text{driving force}}$$

differential operator

Hamiltonian operator \mathcal{J} is **skew-symmetric only for function with compact domain strictly** in Z :

$$\int_a^b \begin{pmatrix} e_1 & e_2 \end{pmatrix} \mathcal{J} \begin{pmatrix} e'_1 \\ e'_2 \end{pmatrix} + \begin{pmatrix} e'_1 & e'_2 \end{pmatrix} \mathcal{J} \begin{pmatrix} e_1 \\ e_2 \end{pmatrix} = - [e_1 e'_2 + e_2 e'_1]_a^b$$

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Power balance

Power balance equation :

$$\begin{aligned}
 \frac{d}{dt} H &= \int_a^b \left(\frac{\delta \mathcal{H}}{\delta \alpha_1} \frac{\partial \alpha_1}{\partial t} + \frac{\delta \mathcal{H}}{\delta \alpha_2} \frac{\partial \alpha_2}{\partial t} \right) dz \\
 &= - \int_a^b \left(\frac{\delta \mathcal{H}}{\delta \alpha_1} \frac{\partial}{\partial z} \frac{\delta \mathcal{H}}{\delta \alpha_2} + \frac{\delta \mathcal{H}}{\delta \alpha_2} \frac{\partial}{\partial z} \frac{\delta \mathcal{H}}{\delta \alpha_1} \right) dz \\
 &= - \left[\frac{\delta \mathcal{H}}{\delta \alpha_1} \frac{\delta \mathcal{H}}{\delta \alpha_2} \right]_a^b
 \end{aligned}$$

If driving forces are zero at the boundary, the total energy is conserved, else there is a flow of power at the boundary. Define two port boundary variables as follows :

$$\begin{pmatrix} f_\partial \\ e_\partial \end{pmatrix} = \begin{pmatrix} \frac{\delta H}{\delta v} \\ \frac{\delta H}{\delta u} \end{pmatrix} \Big|_{a,b}$$

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Port Hamiltonian Formulation

The linear space $\mathcal{D} \ni (f_1, f_2, e_1, e_2, f_\partial, e_\partial)$

- $$\begin{pmatrix} f_1 \\ e_2 \end{pmatrix} = \begin{pmatrix} 0 & -\frac{\partial}{\partial z} \\ -\frac{\partial}{\partial z} & 0 \end{pmatrix} \begin{pmatrix} e_1 \\ e_2 \end{pmatrix}$$
- $$\begin{pmatrix} f_\partial \\ e_\partial \end{pmatrix} = \begin{pmatrix} e_1 \\ e_2 \end{pmatrix} \Big|_{a,b}$$

defines a **Dirac structure**: $\mathcal{D} = \mathcal{D}^\perp$ with respect to the symmetric pairing :

$$\int_a^b e_1 f_1 dz + \int_a^b e_2 f_2 dz + [f_\partial e_\partial]_a^b$$

Port Hamiltonian system

$$\left(\frac{\partial}{\partial t} \alpha, \frac{\delta H}{\delta \alpha}, f_\partial, e_\partial \right) \in \mathcal{D}$$

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Port Hamiltonian Formulation

An other choice for the state variables : $\begin{pmatrix} u \\ v \end{pmatrix}$ the

Hamiltonian system becomes

$$\frac{\partial x}{\partial t} = \begin{pmatrix} 0 & \frac{1}{\mu} \\ \frac{1}{\mu} & 0 \end{pmatrix} \begin{pmatrix} \frac{\delta H}{\delta v} \\ \frac{\delta H}{\delta u} \end{pmatrix}$$

with port variables at the boundary:

$$\begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix} = \begin{pmatrix} v \\ T \frac{\partial u}{\partial z} \end{pmatrix} \Big|_{a,b}$$

which have a non trivial relation with the variational derivatives:

$$\frac{\delta H}{\delta x} = \begin{pmatrix} -\frac{\partial}{\partial z} \left(T \frac{\partial u}{\partial z} \right) \\ \mu v \end{pmatrix}$$

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Generalization of the Port Hamiltonian Systems to infinite dimensional systems

- Canonical formulation of Dirac structures for infinite dimensional systems
- Existence of solution using the semi group theory : the Boundary Control System formulation
- Stabilizing controllers based on the *shaping of the energy and geometric structure*

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General formulation

The system is defined by :

$$f = \mathcal{J}e$$

- Let the **space of flow variables**, \mathcal{F} , and the **space of effort variables**, \mathcal{E} , be real Hilbert spaces.
- Define the space of **bond variables** as $\mathcal{B} = \mathcal{F} \times \mathcal{E}$ endowed by the natural inner product

$$\langle b^1, b^2 \rangle = \langle f^1, f^2 \rangle_{\mathcal{F}} + \langle e^1, e^2 \rangle_{\mathcal{E}}, \quad b^1 = (f^1, e^1), b^2 = (f^2, e^2) \in \mathcal{B}$$

In order to define a Dirac structure, let us moreover endow the bond space \mathcal{B} with a *canonical symmetrical pairing*, i.e., a bilinear form defined as follows:

$$\langle b^1, b^2 \rangle_+ = \langle f^1, r_{\mathcal{E}, \mathcal{F}} e^2 \rangle_{\mathcal{F}} + \langle e^1, r_{\mathcal{F}, \mathcal{E}} f^2 \rangle_{\mathcal{E}}, \quad b^1 = (f^1, e^1), b^2 = (f^2, e^2) \in \mathcal{B}. \quad (1)$$

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Definition on Hilbert space

Denote by \mathcal{D}^\perp the orthogonal subspace to \mathcal{D} with respect to the symmetrical pairing:

$$\mathcal{D}^\perp = \left\{ b \in \mathcal{B} \mid \langle b, b' \rangle_+ = 0 \text{ for all } b' \in \mathcal{D} \right\}. \quad (2)$$

Definition : A Dirac structure \mathcal{D} on the bond space $\mathcal{B} = \mathcal{F} \times \mathcal{E}$ is a subspace of \mathcal{B} which is maximally isotropic with respect to the canonical symmetrical pairing, i.e.,

$$\mathcal{D}^\perp = \mathcal{D}. \quad (3)$$

$$\begin{pmatrix} f \\ e \end{pmatrix} \in \mathcal{D} \iff \text{Power conservation}$$

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Port Hamiltonian System PHS

↪ Definition based on **Dirac structure** and **Hamiltonian function** (total energy of the system).

Definition : Let $\mathcal{B} = \mathcal{E} \times \mathcal{F}$ be the bound space defined above and consider the Dirac structure \mathcal{D} and the Hamiltonian function $\mathcal{H}(x)$ with x the energy variables. Define the flow variables, $f \in \mathcal{F}$ as the time variation of the energy variables and the effort variables $e \in \mathcal{E}$ as the variational derivative of $\mathcal{H}(x)$. The system

$$(f, e) = \left(\frac{\partial x}{\partial t}, \frac{\delta \mathcal{H}}{\delta x} \right) \in \mathcal{D}$$

is a **Port Hamiltonian system** with total energy $\mathcal{H}(x)$

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We start by introducing the differential operator \mathcal{J}

$$\mathcal{J}e = \sum_{i=0}^N P(i) \frac{d^i e}{dz^i}(z) \quad z \in [a, b],$$

where $e \in H^N((a, b); \mathbb{R}^n)$ and $P(i)$, $i = 0, \dots, N$, is a $n \times n$ real matrix with P_N non singular and $P_i = P_i^T (-1)^{i+1}$. Let define

$$Q = \begin{pmatrix} P_1 & P_2 & \cdots & P_N \\ -P_2 & -P_3 & \cdots & 0 \\ \vdots & \cdots & \ddots & \vdots \\ (-1)^{N-1} P_N & 0 & \cdots & 0 \end{pmatrix}$$

Back to the Vibrating string

$$\underbrace{\frac{\partial}{\partial t} \begin{pmatrix} \epsilon \\ p \end{pmatrix}}_f = \underbrace{\begin{pmatrix} 0 & -\frac{\partial}{\partial z} \\ -\frac{\partial}{\partial z} & 0 \end{pmatrix}}_{P_1} \underbrace{\begin{pmatrix} T(z) & 0 \\ 0 & \frac{1}{\mu(z)} \end{pmatrix}}_e \begin{pmatrix} \epsilon \\ p \end{pmatrix}, \quad Q = P_1$$

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↪ We define the **symmetrical pairing** (not depending on \mathcal{J}) of and the **port variables** associated with \mathcal{J} .

Let $\mathcal{F} = \mathcal{E} = L^2((a, b); \mathbb{R}^n) \times \mathbb{R}^{nN}$ and define $\mathcal{B} = \mathcal{F} \times \mathcal{E}$ with the following canonical symmetrical pairing :

$$\begin{aligned} & \langle (f^1, f_{\partial}^1, e^1, e_{\partial}^1) (f^2, f_{\partial}^2, e^2, e_{\partial}^2) \rangle_+ \\ &= \langle e^1, f^2 \rangle_{L^2} + \langle e^2, f^1 \rangle_{L^2} - \langle e_{\partial}^1, f_{\partial}^2 \rangle - \langle e_{\partial}^2, f_{\partial}^1 \rangle, \end{aligned}$$

Definition : The port variables $(e_{\partial}, f_{\partial}) \in \mathbb{R}^{nN}$ associated with \mathcal{J} are defined by :

$$\begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix} = R_{\text{ext}} \begin{pmatrix} e(b) \\ \vdots \\ \frac{d^{N-1}e}{dz^{N-1}}(b) \\ e(a) \\ \vdots \\ \frac{d^{N-1}e}{dz^{N-1}}(a) \end{pmatrix}, \quad R_{\text{ext}} = \frac{1}{\sqrt{2}} \begin{pmatrix} Q & -Q \\ I & I \end{pmatrix}$$

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Theorem : The subspace $\mathcal{D}_{\mathcal{J}}$ of \mathcal{B} defined as

$$\mathcal{D}_{\mathcal{J}} = \left\{ \left(\begin{array}{c} f \\ f_{\partial} \\ e \\ e_{\partial} \end{array} \right) \mid e \in H^N((a, b); \mathbb{R}^n), \mathcal{J}e = f, \left(\begin{array}{c} f_{\partial} \\ e_{\partial} \end{array} \right) = R_{\text{ext}} \left(\begin{array}{c} e(b) \\ \vdots \\ \partial_z^{N-1} e(a) \end{array} \right) \right\}$$

is a Dirac structure, that means that $\mathcal{D} = \mathcal{D}^{\perp}$.

Back to the Vibrating string

$$\underbrace{\frac{\partial}{\partial t} \left(\begin{array}{c} \epsilon \\ p \end{array} \right)}_f = \underbrace{\left(\begin{array}{cc} 0 & -1 \\ -1 & 0 \end{array} \right)}_{P_1} \frac{\partial}{\partial z} \underbrace{\left(\begin{array}{c} T(z)\epsilon \\ \frac{1}{\mu(z)}p \end{array} \right)}_e, Q = P_1$$

$$\left(\begin{array}{c} f_{\partial} \\ e_{\partial} \end{array} \right) = \frac{1}{\sqrt{2}} \left(\begin{array}{cc} P_1 & -P_1 \\ I & I \end{array} \right) \left(\begin{array}{c} e(b) \\ e(a) \end{array} \right) = \frac{1}{\sqrt{2}} \left(\begin{array}{c} T(a)\epsilon(a) - T(b)\epsilon(b) \\ \frac{p(a)}{\mu(a)} - \frac{p(b)}{\mu(b)} \\ T(a)\epsilon(a) + T(b)\epsilon(b) \\ \frac{p(a)}{\mu(a)} + \frac{p(b)}{\mu(b)} \end{array} \right)$$

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Theorem : Let W be a $nN \times 2nN$ full rank matrix. The system

$$\begin{aligned} \dot{x}(t) &= \mathcal{J}e(t) \\ u(t) = \mathcal{B}x(t) &= W \begin{pmatrix} f_{\partial}(t) \\ e_{\partial}(t) \end{pmatrix} \end{aligned}$$

is a **boundary control system**, where $A_W = (\mathcal{J}\mathcal{L})_{ker\mathcal{B}}$ is the **generator of a contraction semigroup** on $L_2((a, b), \mathbb{R}^n)$ if and only if

$$W\Sigma W^T \geq 0 \quad \text{where} \quad \Sigma = \begin{pmatrix} 0 & I \\ I & 0 \end{pmatrix}$$

$$W\Sigma W^T \geq 0 \Leftrightarrow W = S(I + V, I - V)$$

with S invertible and $VV^T \leq I$

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Let define the linear mapping $\mathcal{C} : H^N((a, b), \mathbb{R}^n) \rightarrow \mathbb{R}^{nN}$ as

$$\mathcal{C}x(t) := S_2 \begin{pmatrix} I - V^T & -I - V^T \end{pmatrix} \begin{pmatrix} f_\partial(t) \\ e_\partial(t) \end{pmatrix}$$

and the output as $y(t) = \mathcal{C}x(t)$, then for $u \in C^2((0, \infty); \mathbb{R}^{nN})$ and $x(0) - Bu(0) \in D(J_W)$ the following balance equation is satisfied:

$$\frac{1}{2} \frac{d}{dt} \|x(t)\|^2 = \begin{pmatrix} u^T(t) & y^T(t) \end{pmatrix} P_W \begin{pmatrix} u(t) \\ y(t) \end{pmatrix}.$$

where $P_W = \begin{pmatrix} \frac{1}{4} S^{-T} (P_1^2 - P_1 V V^T P_1) S^{-1} & -2 S^{-T} P_1 V P_2 S^T \\ -2 S P_2 V^T P_1 S^{-1} & 4 S (-P_2^2 + P_2 V^T V P_2) S^T \end{pmatrix},$

$$P_1 = (I + V V^T)^{-1}, \quad P_2 = (I + V^T V)^{-1}.$$

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Particular cases

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$$V = 0 \begin{cases} \dot{x}(t) &= \mathcal{J}x(t), \\ u(t) &= \frac{1}{2} (f_{\partial}(t) + e_{\partial}(t)) \\ y(t) &= \frac{1}{2} (f_{\partial}(t) - e_{\partial}(t)) \end{cases} \implies$$

boundary control system, with the associated semigroup a contraction

$$\frac{1}{2} \frac{d}{dt} \|x(t)\|^2 = \|u(t)\|^2 - \|y(t)\|^2$$

$$V = I \begin{cases} \dot{x}(t) &= \mathcal{J}x(t) \\ u(t) &= f_{\partial}(t) \\ y(t) &= -e_{\partial}(t) \end{cases} \implies$$

boundary control system, with the associated semigroup unitary

$$\frac{1}{2} \frac{d}{dt} \|x(t)\|^2 = u(t)^T y(t)$$

Vibrating string :

$$V = 0 \implies u = \frac{1}{\sqrt{2}} \begin{pmatrix} T(a)\epsilon(a) \\ \frac{p(a)}{\mu(a)} \end{pmatrix} \text{ and } y = \frac{1}{\sqrt{2}} \begin{pmatrix} -T(b)\epsilon(b) \\ -\frac{p(b)}{\mu(b)} \end{pmatrix}$$

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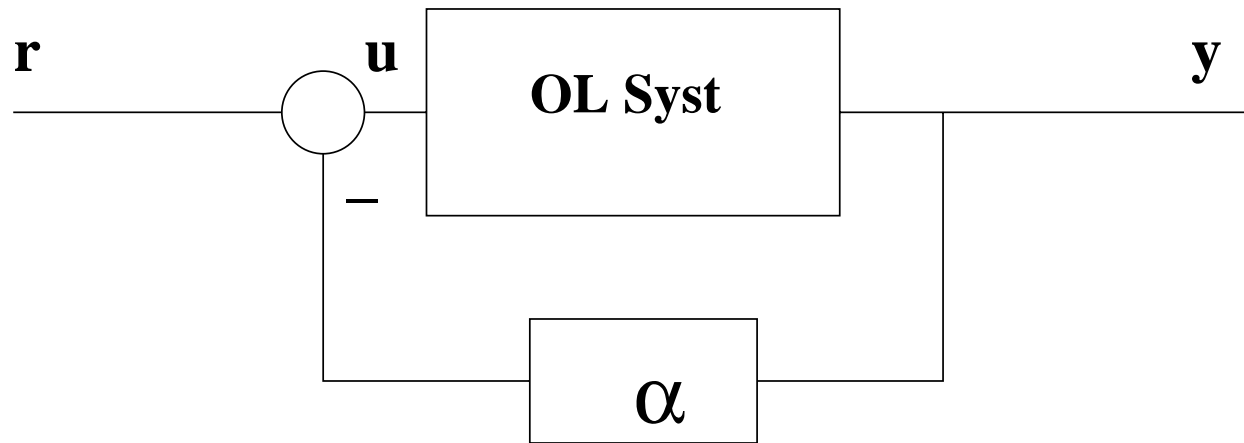
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- Stability of IPS
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Static feedback for IPS



Open loop system

$$\dot{x} = \mathcal{J}x$$

$$u = W_{imp} \begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix}$$

$$y = C_{imp} \begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix}$$

Closed loop system

$$\dot{x} = \mathcal{J}x$$

$$r = (W_{imp} + \alpha C_{imp}) \begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix}$$

$$y = C_{imp} \begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix}$$

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Lemma : The closed loop system described by

$$\dot{x} = \mathcal{J}_{\mathcal{L}}x$$

$$r = (W_{imp} + \alpha C_{imp}) \begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix}$$

$$y = C_{imp} \begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix}$$

is a **boundary control system**. Furthermore, the operator $A_s = \mathcal{J}_{\mathcal{L}}|_{D(A_s)}$ generates a **contraction semigroup** on $X = L_2((a, b); \mathbb{R}^n)$ where

$$D(A_s) = \left\{ x \in D(\mathcal{J}) \mid \begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix} \in \ker \tilde{W} \right\}$$

and $\tilde{W} = (W_{imp} + \alpha C_{imp})$ is a full rank $nN \times 2nN$ matrix

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Theorem : $(\lambda - A_s)^{-1} : X \rightarrow X$ is a compact operator for $\lambda > 0$. Then the system described by $(VV^T = 0)$:

$$\dot{x} = \mathcal{J}x$$

$$r = (W_{imp} + \alpha C_{imp}) \begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix}$$

$$y = C_{imp} \begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix}$$

with $r = 0$ is **globally asymptotically stable**. For any $x(0) \in X$ the unique (classical or weak) solution $x(t) = T(t)x(0)$ of the closed loop system asymptotically approaches to zero, i.e.

$$\lim_{\infty} \|x(t)\|_X = 0$$

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Theorem : If $W\Sigma W^T > 0$ the system described by :

$$\begin{aligned} \dot{x} &= \mathcal{J}x \\ r &= W \begin{pmatrix} f_\partial \\ e_\partial \end{pmatrix} \\ y &= C \begin{pmatrix} f_\partial \\ e_\partial \end{pmatrix} \end{aligned}$$

with $r = 0$ is **globally asymptotically stable**. For any $x(0) \in X$ the unique (classical or weak) solution $x(t) = T(t)x(0)$ of the closed loop system asymptotically approaches to zero, i.e.

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~> Port hamiltonian formulation of Distributed parameter systems arises from two adjoint relations :

- conservation law
- generating forces being variational derivative of potential

and use of Stokes Theorem.

~> Underline the structural properties

~> Asymptotic stability can be checked by looking a condition on a matrix.

~> Links with well posed, conservative, boundary control systems.

~> Generalization to other operators using feedback ideas.

Future work :

~> Exponential stability ...

~> Controller design.

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Introduction

Dirac Structure
and PHS defined
on Hilbert Space

Boundary Control
Systems (BCS)

Closed loop BCS,
stability etc ...

Conclusion and
further works

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